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METHODS AND APPARATUS FOR MANUFACTURING FIBER-CEMENT SOFFITS WITH AIR VENTS

TECHNICAL FIELD

The present invention relates to construction materials to protect the exterior of houses and other structures. More particularly, the present invention relates to fiber-cement soffits for installation under the eaves of houses, commercial buildings and other structures.

BACKGROUND

A significant portion of the construction industry builds residential and commercial structures. Contractors generally build structures in-situ at specific sites, and "manufactured builders" generally build sections of structures in a factory for assembly at a particular site. In either application, the structures are generally framed, roofed and then covered with exterior siding materials. One particularly advantageous and popular type of siding is fiber-cement siding. Fiber-cement siding products are typically made from a composition having cement, cellulosic materials and a binder. The fiber-cement composition is pressed, cured and then cut into panels, shakes and planks to form finished siding products that are ready to be installed on a structure. Fiber-cement siding products are insect resistant, fire resistant, and wear resistant. Fiber-cement siding products can also be painted like wood, but they are not made from a valuable natural resource. Therefore, many contractors and manufactured builders are switching to fiber-cement siding products from wood, composites, aluminum, plastic and bricks.

Several buildings also have soffits installed under the eaves where the roof overhangs the exterior walls. Soffits are conventionally made from wood, metal (aluminum) or plastics. Soffits typically have large holes that are covered with a large mesh screen or thin slots to provide ventilation and to prevent insects or birds from nesting within the structure. The large holes, for example, are generally 1.5-3.0 inch diameter circles or 2 x 12 inch rectangles that are cut with a jig saw or a cylindrical saw. Wood and wood composite soffits, however, have several drawbacks because they are subject to insect infestation,

warping, rotting and fire. Aluminum and plastic soffits also have drawbacks because they are difficult to paint, and thus the color of the soffits may be substantially different than the color of the paint on the exterior siding. Therefore, because fiber-cement building products do not suffer from the same drawbacks as wood, plastic or aluminum building products. many contractors and manufactured builders would like to install soffits made from fiber-cement.

Manufacturing fiber-cement products, however, can be difficult because fiber-cement building products are more difficult to process than wood, plastics or aluminum. For example, cutting fiber-cement products with circular saws (e.g., a rotating abrasive disk) produces a significant amount of dust that makes the working environment unpleasant and difficult to clean. Fiber-cement building products are also relatively brittle and can easily crack during processing. Moreover, fiber-cement building products are much more abrasive than wood, plastics or aluminum, and thus they wear through cutting tools very quickly. Fiber-cement soffits are particularly difficult to manufacture because it is difficult and time-consuming to form apertures in fiber-cement panels that allow air to flow through the soffits. Thus, fiber-cement soffits are not yet widely used in the marketplace.

One particularly promising fiber-cement soffit is a 12-foot fiber-cement panel having a plurality of 1/8 inch diameter apertures in a uniform, symmetrical pattern. Manufacturers of fiber-cement building products, such as James Hardy Building Products of Fontana, California, have experimented with manufacturing such fiber-cement soffits by drilling the apertures. Drilling the fiber-cement panel, however, is not generally feasible in large scale production because it is too time-consuming and the abrasive fiber-cement quickly wears down the drill bits. Drilling the fiber-cement panel also produces a fine dust that is unpleasant and difficult to clean. Therefore, drilling the apertures in the fiber-cement panel is not a viable manufacturing process.

To overcome the problems of drilling fiber-cement panels, manufacturers of fiber-cement building products have also experimented with punching individual holes

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through a fiber-cement panel using a sheet metal punch. Typical sheet metal punches have a very small clearance between the punch and the die. Punching apertures through the fiber-cement panel with a sheet metal punch is also not feasible because the sheet punch metal often sticks to the fiber-cement panel. The sheet metal punch may thus delaminate portions of the panel as it withdraws from the aperture. Punching apertures through the fiber-cement panel with a sheet metal punch may also produce a mushroom-shaped plug such that each aperture has a small opening on the front side but a much larger opening on the back side. In preliminary tests using a sheet metal punch to form apertures in a fiber-cement panel, the sheet metal punch ripped out so much material from the backside of the panel that a typical 12-foot soffit may not have sufficient structural integrity to be hung under the eaves of a structure.

SUMMARY OF THE INVENTION

The present invention is directed toward methods and apparatuses for producing fiber-cement soffit building products. In one embodiment of the invention, an apparatus for producing fiber-cement soffits includes a punch assembly, a support assembly facing at least a portion of the punch assembly, and an actuator operatively coupled to at least one of the punch assembly or the support assembly. The punch assembly can include a punch plate and a plurality of punches coupled to the punch plate. Each punch can have a length and a first cross-sectional dimension generally normal to the length. The support assembly can have a support plate, and at least a portion of the support plate is juxtaposed to at least a portion of the punch plate. The support plate can include a plurality of holes arranged in a pattern so that each hole in the portion of the support plate juxtaposed to the punch plate is aligned with a corresponding punch on the punch plate. Each hole can have a second cross-sectional dimension greater than the first cross-sectional dimension of the punches to define a radial punch/hole clearance between each punch and each hole. The radial punch/hole clearance, for example, is generally greater than that of metal punch presses to allow the punches to be removed from a fiber-cement panel without delaminating portions of the panel.

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The actuator can be coupled to the punch plate to move the punches between a first position and a second position. In the first position, the punches are spaced apart from the support plate to allow a fiber-cement panel to pass between the punches and the support plate. In the second position, the punches penetrate into the fiber-cement panel to form a plurality of apertures in the fiber-cement panel. The apertures generally have a first opening on a front side of the panel facing the punches and a second opening on the backside of the panel facing the support plate. The first openings can have shapes corresponding to the first cross-sectional dimension of the punches, and the second openings are slightly larger than the first openings. The apertures are thus frustoconical with only a slight change in diameter from the top to the bottom.

The punch and support assemblies can have several different configurations. In one particular embodiment, the punch plate is a first flat plate and the support plate is a second flat plate. Other embodiments of the punch plate and support plate include first and second cylindrical members, or devices having other shapes that can be pressed together. The punches coupled to the punch plate and the holes in the support plate can also have several configurations. In one particular embodiment, the punches have a concave contact face and a first diameter defining the first cross-sectional dimension. The first diameter, for example, can be approximately 0.11-0.25 inch. The holes in the support plate of this embodiment have a second diameter defining the second cross-sectional dimension. The second diameter can be approximately 0.18-0.39 inches. The radial punch/hole clearance between the punches and the holes in these particular embodiments can accordingly be approximately 0.032-0.070 inch. The radial punch/hole clearance can also be a function of the thickness of the fiber-cement panel or the size of the punch. For example, the radial punch/hole clearance between the punches and the holes can be approximately 4%-40% of the thickness of the fiber-cement panel or approximately 4%-30% of the diameter of the holes.

In one particular embodiment, the punch assembly includes a plurality of punches having a concave contact face, a first diameter of approximately 0.115-0.135 inch,

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and a biasing element surrounding each punch. The support plate of this particular embodiment can have holes with a second diameter of approximately 0.150-0.250 inch.

In the operation of this particular embodiment, the actuator drives the punch assembly toward the support plate until the punches penetrate through only a portion of the fiber-cement panel. The punches accordingly do not pass completely through the panel in this embodiment. Although the punches penetrate the fiber-cement panel only to an intermediate depth, the punches remove a frustoconical shaped plug from the panel to produce apertures through the full thickness of the fiber-cement panel. The biasing elements also press against the panel to prevent the panel from sticking to the punches as the punches withdraw from the fiber-cement panel. In this particular embodiment, the radial punch/hole clearance and the biasing elements prevent the punches from sticking to the fiber-cement panel to avoid or prevent delamination of the fiber-cement at the apertures.

BRIEF DESCRIPTION OF THE DRAWINGS

Figure 1 is a top plan view of a punch press for producing a fiber-cement panel in accordance with one embodiment of the invention.

Figure 2 is a cross-sectional side view of the punch press of Figure 1 taken along line 2-2.

Figures 3A and 3B are partial cross-sectional views of a punch assembly and a support assembly of a punch press in accordance with one embodiment of the invention for producing a fiber-cement soffit from a fiber-cement panel.

Figures 4A and 4B are side elevation views of punches in accordance with particular embodiments of the invention.

Figure 5 is a schematic cross-sectional view of another punch press for producing a fiber-cement soffit from a fiber-cement panel in accordance with another embodiment of the present invention.

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Figures 6A and 6B are partial cross-sectional views of still another punch press for producing a fiber-cement soffit from a fiber-cement panel in accordance with still another embodiment of the invention.

DETAILED DESCRIPTION OF THE INVENTION

The present invention is a method and apparatus for fabricating fiber-cement soffits and other building materials from fiber-cement panels. Several specific details of the invention are set forth in the following description and in Figures 1-6B to provide a thorough understanding of certain embodiments of the present invention. The following description also provides examples of the preferred embodiments of the invention. One skilled in the art, however, will understand that the present invention may have additional embodiments, or that other embodiments of the invention may be practiced without several of the specific features explained in the following description.

Figure 1 is a top plan view and Figure 2 is a side elevation view of a punch press 10 for producing a fiber-cement soffit 12 from a panel 14 of fiber-cement. The panel 14 is made from cement, a cellulosic material and a binder. James Hardy Building Products of Fontana, California produces the panel 14 without any holes. The panel 14 is typically 4-48 inches wide, 8-16 feet long, and 0.25-0.625 inch thick. The punch press 10 produces the fiber-cement soffit 12 by forming a plurality of apertures 16 in the panel 14 without delaminating the panel 14 or removing an excessive amount of material from the backside of the panel 14.

As best shown in Figure 2, the punch press 10 can have a support structure 20 with a support surface 21, an upper frame 22 and a lower frame 24. In this embodiment, the punch press 10 includes a punch assembly 40 having a punch plate 42 and a plurality of punches 50 coupled to the punch plate 42. Each punch 50 has length projecting downward from the punch plate 42 and a first cross-sectional dimension in a plane normal to the length. The first cross-sectional dimension can be circular, rectilinear, or any other suitable shape. The punches 50 are generally made from metal, ceramic, or other hard materials. The punch

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assembly 40 can also include a plurality of biasing elements 51 that press against the panel 14 when the punches 50 penetrate the panel 14. A biasing element 51 can be adjacent to each punch 50, or fewer biasing elements can be attached to the punch assembly 40. The biasing elements 51, for example, can be springs, compressible and resilient tubes made from rubber or another resilient material, or other types of compressible and resilient members.

The punch press 10 can also include a support assembly 60 having a support plate 62 with a plurality of holes 64. At least a portion of the support plate 62 is juxtaposed to the punch plate 42. The holes 64 can be arranged in the same pattern as the punches 50. When the punch plate 42 and the support plate 62 are flat plates, each hole 64 is generally aligned with a corresponding punch 50. Each hole 64 also has a second cross-sectional dimension greater than the first cross-sectional dimension of a corresponding punch 50 to provide a radial punch/hole clearance between each punch 50 and each hole 64. The radial punch/hole clearance is sufficient to allow the punches to be removed from the panel 14 without delaminating portions of the panel 14.

The punch press 10 further includes an actuator 70 that can be coupled to either the punch assembly 40 or the support assembly 60 to move the punches 50 and/or the support plate 62 toward one another. In the embodiment shown in Figures 1 and 2, the actuator 70 is attached to the upper frame 22, and the punch plate 42 is attached to actuator 70. The actuator 70 reciprocates the punch plate 42 and the punches 50 along a punch stroke "P" and a retraction stroke "R." The actuator 70 can be a hydraulic or pneumatic actuator that quickly drives the punch plate 42 along the punch stroke P, and then retracts the punch plate 42 along the retraction stroke R. Suitable actuators 70 for the punch press 10 are manufactured by Rouselle Press Company of Chicago, Illinois. The punch press 10 generally operates by indexing the panel 14 across the support plate 62 (arrow I), and then reciprocating the punch plate 42 along the punch stroke P and the retraction stroke R to punch the apertures 16 in incremental sections of the panel 14.

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Figures 3A and 3B are partial cross-sectional views of the punch assembly 40 and the support assembly 60 in accordance with a particular embodiment of the punch press 10. Referring to Figure 3A, the punch plate 42 is a rigid, flat plate. The punches 50 can be threadedly attached to the punch plate 42, and have a first diameter d₁ defining the first cross-sectional dimension. Each punch 50 can also have a concave contact face 57 and a sharp rim 58. A cylindrical biasing element 51 is threadedly attached to each punch 50. In this embodiment, the biasing elements 51 are polymeric tubes or sleeves that have lengths approximately equal to or exactly equal to the lengths of the punches 50.

The support plate 62 of Figure 3A is also a flat, rigid plate that faces the punch plate 42. The holes 64 in the support plate 62 have a second diameter d₂ defining the second cross-sectional dimension. The second diameter d₂ is greater than the first diameter d₁ to provide the radial punch/hole clearance "C" between the punches 50 and the holes 64. The first diameter d₁ of the punches 50 and the second diameter d₂ of the holes 64 can be a function of the thickness of the panel 14, the size of the apertures 16, or other parameters. For a panel 14 having a thickness T of 0.25-0.3125 inch, the first diameter d₁ of the punches 50 can be approximately 0.11-0.25 inch, and the second diameter d₂ of the holes 64 can be approximately 0.18-0.39 inch. The clearance C between the punches 50 and the holes 64 for such a panel 14 can be approximately 0.032-0.125 inch, and more preferably approximately 0.04-0.07 inch. In further applications of the punch press 10, the radial punch/hole clearance C can be approximately 23%-27% of the second diameter d₂. In still further applications of the punch press 10, the radial punch/hole clearance C is approximately 4%-40% of the thickness T of the panel 14, and more preferably 18%-27% of the thickness T.

In one particular embodiment of the punch press 10, the punches 50 initially have a first diameter d_1 of approximately 0.135 inch and the holes 64 have a second diameter d_2 of approximately 0.25 inch. The initial radial punch/hole clearance C is 0.0575 inch, or approximately 23% of the second diameter d_2 . The fiber-cement composition of the panel 14, however, wears down the punches 50 such that the diameter d_1 of a shank portion of the

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punches 50 decreases. The diameter of the rim 58 of a punch 50 with a concave contact face 57 generally does not decrease as much as the shank, and thus the size of the apertures 16 do not decrease significantly as the shank of the punch wears down. When the diameter of the shank of the punch 50 is approximately 0.115 inch, the radial punch/hole clearance at the shank can be approximately 0.0675 inch. It is recommended that the punches 50 be replaced when the shank portions have a diameter of approximately 0.115 inch to avoid breakage of the punches 50.

Figure 3B illustrates several aspects of operating the embodiment of the punch assembly 40 shown in Figure 3A. The actuator 70 (Figures 1 and 2) drives the punch assembly 40 toward the support assembly 60 so that the punches 50 penetrate into the panel 14. In a typical application, the punches 50 do not pass completely through the panel 14, but rather the punches 50 stop at an intermediate depth D_i in the panel 14. The intermediate depth D_i is approximately 0.0625-0.1875 inch for a 0.25-0.31625 inch thick panel 14. In other embodiments of operating the punch assembly 40, the actuator 70 drives the punches 50 completely through the fiber-cement panel 14. As the punches 50 penetrate to the intermediate depth D₁, the fiber-cement panel 14 fractures along approximately conical paths to eject frustoconical plugs 18 from the fiber-cement panel 14. Each punch 50 accordingly forms an aperture 16 having a well-defined opening 16 at a front side of the soffit 12 facing the punch plate 42 and a slighter rougher opening 16 at a backside facing the support plate 62. The actuator 70 then retracts the punch assembly 40 to withdraw the punches 50 from the soffit 12. As the punches 50 withdraw from the soffit 12, the biasing elements 51 push the soffit 12 toward the support plate 62 to prevent the soffit 12 from sticking to the punches 50.

The particular embodiments of the punch press 10 shown in Figures 1-3B quickly produce large volumes of finished fiber-cement soffit. One feature of the punch press 10 is that the actuator 70 (Figures 1 and 2) can quickly reciprocate the punch assembly 40 along the punch stroke P and retraction stroke R (Figure 2) to punch the apertures 16 through the desired length of the panel 14 in a matter of seconds. The embodiments of the

punch press 10 shown in Figures 1-3B, therefore, can produce a high volume of finished fiber-cement soffit 12 in a short period of time with relatively inexpensive equipment.

The embodiments of the punch press 10 in Figures 1-3B produce the finished soffit 12 without producing noticeable amounts of dust or other small particulate matter. Unlike drills that produce small particles to form holes in the panel 14, the punch press 10 produces plugs 18 that fall to the floor and do not become an airborne contaminate. The punch press 10, accordingly, is not only easy to operate, but it also provides a clean, dust-free environment.

Another feature of the embodiments of the punch press 10 shown in Figures 1-3B is that they produce well-defined holes at both the front side and the backside of the soffit 12. In contrast to metal punches that have very tight tolerances between the punches and the dies (e.g., generally 0.03125 inch or less for 0.25-0.3125 inch thick metal sheets), the larger radial punch/hole clearance C between the punches 50 and the holes 64 reduces the size of the opening 16 (Figure 3B) at the back side of the soffit 12. The resulting soffit produced with the embodiments of the punch press 10 shown in Figures 1-3B accordingly has good structural integrity compared to fiber-cement panels that have been punched with metal punch presses having much smaller radial punch/hole clearances.

The embodiments of the punch press 10 shown in Figures 1-3B also produce a fiber-cement soffit 12 in which the material at the apertures 16 does not delaminate. The radial punch/hole clearance C between the punches 50 and the holes 64 is large enough to reduce binding between the punches 50 and the fiber-cement panel 14. Additionally, the biasing elements 51 press against the fiber-cement panel 14 adjacent to the punches 50. As such, the combination of the downward force applied by the biasing elements 51 and the reduced friction between the punches 50 and the panel 14 allows the punches 50 to withdraw from the panel 14 without delaminating the fiber-cement material adjacent to the punches 50. Therefore, the embodiments of the punch press 10 shown in Figures 1-3B are expected to produce an extremely durable fiber-cement soffit 12.

Figures 4A and 4B are side elevation views of different punches in accordance with particular embodiments of the invention. Referring to Figure 4A, a punch 50 like the ones illustrated in Figures 3A and 3B is shown in more detail. The punch 50 can have a threaded section 53 to threadedly attach the punch 50 to the punch plate 42. The punch 50 can also have a punch section 54 with the first diameter d₁, a concave contact face 57, and a rim 58. The concave face 57 and the rim 58 are expected to provide better directional control of crack propagation through the panel 14 so that the difference between the opening 16a and the opening 16b is not significant. Referring to Figure 4B, a punch 50 has a threaded section 53 and punch section 54 with a flat contact face 57a.

Figure 5 is a schematic side elevation view of a punch press 100 that includes an indexing and control system in accordance with another embodiment of the invention. The punch press 100 can include a punch assembly 40, a support assembly 60 and actuator 70 similar to those described above with reference to Figures 1-4B. The actuator 70 can be attached to an upper frame 122, and the punch assembly 40 can be coupled to the actuator 70. The support assembly 60 can be coupled to the support structure 120 so that the support assembly 60 is juxtaposed to the punch assembly 40.

The punch press 100 can also include a first passive roller support array 170a on the feed side of the punch assembly 40, and a second passive roller array 170b on a discharge side of the punch assembly 40. The first passive roller array 170a generally includes a plurality of passive rollers 171a coupled to a frame 172a, and the second passive roller array 170b includes a plurality of second passive rollers 171b coupled to a second frame 172b. The first and second passive rollers 171a and 171b are positioned so that the upper apex of each passive roller is at an elevation at least proximate to the elevation of the support plate 62.

The punch press 100 also includes a first active roller assembly 174 between the first passive roller array 170a and the punch assembly 40, and a second active roller assembly 176 between the punch assembly 40 and the second passive roller array 170b. The

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first active roller assembly 174 initially moves the panel 14 into position under the punch assembly 40 and then incrementally feeds the panel 14 across the support assembly 60. The second active roller assembly 176 also feeds the panel 14 across the support assembly 60 and then discharges a finished fiber-cement soffit (not shown in Figure 5) across the second passive roller array 170b.

The punch press 100 also includes a control system to coordinate the indexing of the panel 14 and the operation of the actuator 70 to incrementally punch apertures 16 (Figure 3B) through portions of the panel 14. The control system can include a first position sensor 182 to sense a leading edge 15a of the panel 14, and a second position sensor 184 to sense a trailing edge 15b of the panel 14. The first position sensor 182 is preferably an optical sensor positioned between the first active roller assembly 174 and the punch assembly 40. The second position sensor 184 is preferably an optical sensor positioned between the punch assembly 40 and the second active roller assembly 176. The control system further includes a controller 190 coupled to the actuator 70, the first and second active roller assemblies 172 and 174, and the first and second position sensors 182 and 184. Suitable controllers for operating the punch press 10 are available from Rouselle Press Company.

The operation of the punch press 100 will now be described. The first active roller assembly 174 initially rotates at a relatively low rotational velocity to draw the panel 14 towards the punch assembly 40 until the leading edge 15a is aligned with the first position, sensor 182. The first position sensor 182 sends a signal to the controller 190 indicating the location of the leading edge 15a, and the controller 190 resets the punch press 100 for a new cycle by confirming that the punch assembly 40 is in a raised position and by stopping the rotation of the first active roller assembly 174. The controller 190 then signals the first and second active roller assemblies 174 and 176 to rotate at a relatively high velocity for an initial incrementing distance to position a first section 17a of the panel 14 between the punch assembly 40 and the support assembly 60. The controller 190 stops the rotation of the first and second active roller assemblies 174 and 176 when the first section 17a of the panel 14 is

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in place. The controller 190 then initiates the punch stroke of the actuator 70 to drive the punches 50 into the first section 17a of the panel 14 and the retraction stroke of the actuator 70 to withdraw the punches 50 from the panel 14. The controller 190 subsequently initiates the first and second active roller assemblies 174 and 176 to move the panel 14 until a second section 17b of the panel 14 is aligned with the punch assembly 40 and the support assembly 60. The controller 190 repeats this operation until apertures are formed along a desired length of the panel. As the trailing edge 15b of the panel 14 passes underneath the second position sensor 184, this sensor sends a signal to the controller 190 that the punch press 100 is clear and ready for processing another panel 14. The second position sensor 184 accordingly prevents another panel 14 from being fed through the first active roller assembly 174 while another panel 14 is still under the punch assembly 40 to prevent damaging the punches 50 or jamming the punch press 100.

Figures 6A and 6B are schematic cross-sectional views of a punch press 200 in accordance with still another embodiment of the invention. Referring to Figure 6A, the punch press 200 includes a support structure 220 having an upper frame 222 and a lower frame 224. The support structure 220 further includes a first passive roller assembly 126a having a plurality of passive rollers 127a, and a second passive roller assembly 126b having a plurality of second passive rollers 127b.

The punch press 200 also includes a punch assembly 240 and a support assembly 260. In this embodiment, the punch assembly 240 has a cylindrical punch plate 242 with a plurality of punch cavities 244 spaced radially apart from one another around the circumference of the punch plate 242. The cavities 244 can also extend in rows along an axial length of the cylindrical punch plate 242. The punch plate 242 has an end panel 249 or spokes attached to a ring bearing 225 on the upper frame 222 to rotatably attach the punch plate 242 to the support structure 220. The punch plate 242 can be driven by an active roller 248 attached to the upper frame 222. The support assembly 260 of this embodiment has a cylindrical support plate 262 rotatably attached to the lower frame 224 at a hub 265 by a

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number of spokes 266. The support assembly 260 can also include a drive roller 263 attached to the lower frame 224.

The punch press 200 further includes an actuator 270 attached to the upper frame 222 inside of the ring bearing 225. The actuator 270 has a ram 272 located within the cylindrical punch plate 242. The ram 272, for example, can be a plate extending along the axial length of the cylindrical plate 242.

Figure 6B illustrates the punch assembly 240 and the support assembly 260 in further detail. The punch assembly 240 further includes a plurality of punches 250 received in the punch cavities 244 of the cylindrical punch plate 242. The punches 250 are preferably arranged in rows such that a row of punches 250 extends along the axial length of the cylindrical punch plate 242 at each radial position R₁, R₂, etc. Each punch cavity 244 has a first section 245 at the outer surface of the punch plate 242, a second section 246 having a larger cross-sectional than the first section 245, and a third section 247 with a smaller cross-section than the second section 246. The punch 250 has a punch section 246 of the cavity 244, and a head 253 passing through the third section 247 of the cavity 244. Each punch 250 also has a biasing element 255 between the shoulder 252 and an outer rim of the second section 246 of the cavity 244 defined by the difference between the diameters of the first section 245 and the second section 246.

The punch press 200 operates by driving the ram 272 against the heads 253 of a row of punches 250 under the ram 272. The row of punches 250 and the row of holes 264 aligned with the ram 272 define an active punch set in a punch position. The actuator 70 then retracts the ram 272 so that the biasing elements 255 push the punches 250 toward the interior of the punch plate 242. The biasing elements 255 hold the shoulders 252 of the punches 250 against an inner rim defined by the difference between the diameters of the second section 246 and the third section 247 of the punch cavity 244 (shown as a passive punch set at radial location R₂ in Figure 6B). The drive motors 248 and 263 can

continuously rotate the punch plate 242 and the support plate 260 as the actuator 270 reciprocates the ram 272 to continuously punch apertures through the fiber-cement panel 14.

Although the foregoing sets forth specific embodiments of the invention, it will be appreciated that various modifications may be made to the specific embodiments described above without deviating from the spirit and scope of the invention. For example, the punch assembly 40 and the support assembly 60 can extend along the full length of the panel 14 so that all of the apertures 16 can be punched in one stroke of the actuator 70. Additionally, the apparatus and process can be used to punch holes in fiber-cement panels having diameters larger than 0.25 inch (e.g., 1.0-3.0 inches) with a radial punch/hole clearance of approximately 0.032-0.070 inch. Such large holes can then be covered with a mesh or screen to keep insects and birds out of protected spaces. The specific embodiments described above provide sufficient information to enable a person skilled in the art to make and use the best modes of the invention, but the claims are not limited to the particular embodiments described above. Accordingly, the invention is not limited except as by the appended claims.